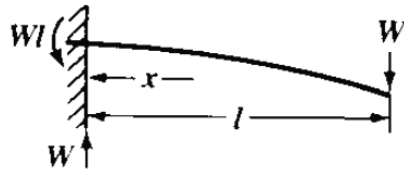


Case 1: constant drill pipe



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Modulus of Elasticity

$$E := 30 \cdot 10^6 \text{ psi}$$

Load

$$W := -1000 \text{ lbf}$$

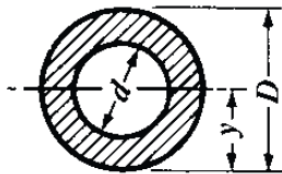
Some distance as indicated

$$x := 120 \text{ in}$$

Some distance as indicated

$$L := 120 \text{ in}$$

Finding Moment of Inertia:



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Outside Diameter

$$D := \left(6 + \frac{5}{8} \right) \text{ in}$$

Inside Diameter

$$d := 5.901 \text{ in}$$

Moment of Inertia

$$I := \frac{\pi(D^4 - d^4)}{64} = 35.04 \cdot \text{in}^4$$

section modulus of the cross-section of the beam

$$Z := \pi \frac{(D^4 - d^4)}{32 \cdot D} = 10.578 \cdot \text{in}^3$$

Stress at x

$$s_x := \frac{W}{Z}(L - x) = 0 \cdot \text{ksi}$$

Stress at mount

$$s_L := \frac{(W \cdot L)}{Z} = -11.344 \cdot \text{ksi}$$

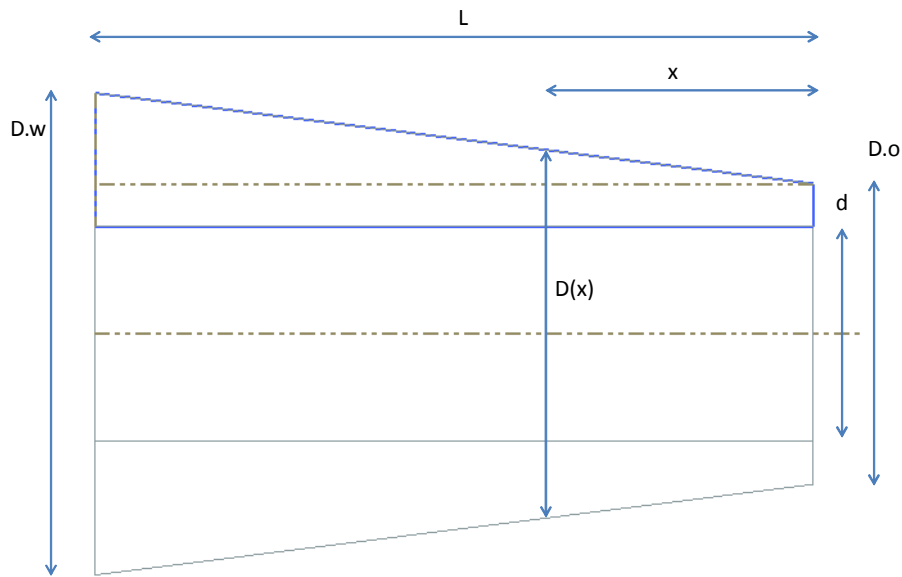
Deflection

$$y_1 := \frac{W \cdot x^2}{6E \cdot I}(3 \cdot L - x) = -0.548 \cdot \text{in}$$

Case 2: Tapered Deflection

$$D_w := 6.625\text{in} \quad D_o := 6.625\text{in} \quad d := 5.901\text{in} \quad L := 120\text{in} \quad W := -1000\text{lbf} \quad E := 30 \cdot 10^6 \cdot \text{psi}$$

$$x := 0..120$$



$$D(x) := \left[(D_w - D_o) \cdot \frac{(x)}{L} \right] + D_o$$

$$D(0\text{in}) = 6.625 \cdot \text{in}$$

$$D(120\text{in}) = 6.625 \cdot \text{in}$$

$$M(x) := W \cdot (x)$$

$$M(0 \cdot \text{in}) = 0 \cdot \text{lbf} \cdot \text{in}$$

$$M(120\text{in}) = -120000 \cdot \text{lbf} \cdot \text{in}$$

$$I(x) := \frac{\pi}{64} \cdot (D(x)^4 - d^4)$$

$$I(0 \cdot \text{in}) = 35.04 \cdot \text{in}^4$$

$$I(120 \cdot \text{in}) = 35 \cdot \text{in}^4$$

$$\theta(x) := \left(\int_0^x \frac{M(x)}{E \cdot I(x)} dx \right)$$

$$\theta(0\text{in}) = 0 \cdot \text{deg}$$

$$\theta(120\text{in}) = -0.392 \cdot \text{deg}$$

$$y(x) := \int_0^L \int_0^x \frac{M(x)}{E \cdot I(x)} dx dx - (x) \cdot \left(\int_0^x \frac{M(x)}{E \cdot I(x)} dx \right)$$

$$y(0 \cdot \text{in}) = -0.274 \cdot \text{in} \quad y(120 \cdot \text{in}) = 0.548 \cdot \text{in}$$

